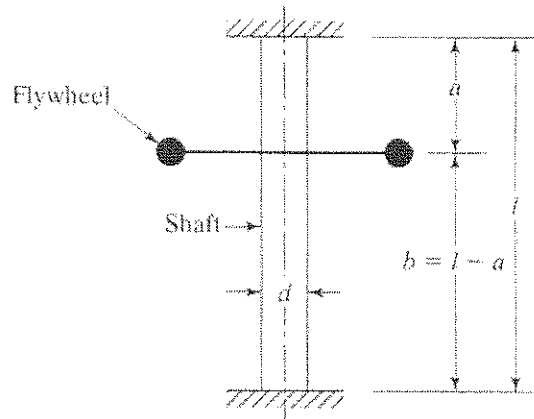


## ME 440 Intermediate Vibrations

Homework #3  
due Thursday, October 5, 2017

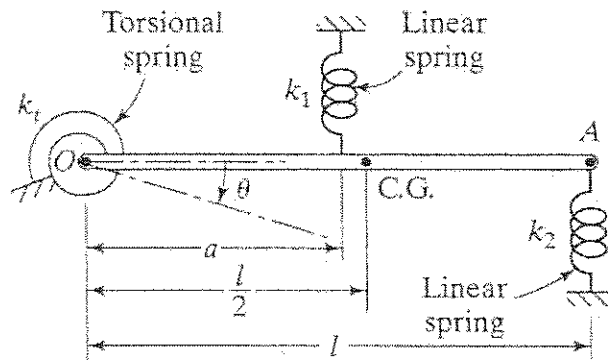
### Problem 1

A flywheel is mounted on a vertical shaft, as shown below. The shaft has a diameter  $d$  and length  $l$  and is fixed at both ends. The flywheel has a weight of  $W$  and a radius of gyration of  $r$ . Find the natural frequency of the longitudinal, the transverse, and the torsional vibration of the system.



### Problem 2

The uniform rigid bar  $OA$  of length  $L$  and mass  $m$  is pinned about point  $O$ . Using Newton's Second Law, find the equation of motion for the system using the generalized coordinate  $\theta$  and also find the system's natural frequency.



TRANSVERSE (NO ROTATION)  
(OPTION 1)

FIXED - FIXED BEAM WITH  
LATERAL DISPLACEMENT

$$k = \frac{12EI}{l^3}$$

$$k_a = \frac{12EI}{a^3}$$

$$k_b = \frac{12EI}{b^3}$$

BOTH BEAMS HAVE THE SAME DEFLECTION/DEFORMATION  
°° PARTIAL

$$k_{eq} = k_a + k_b$$

$$= \frac{12EI}{a^3} + \frac{12EI}{b^3} = 12EI \left( \frac{1}{a^3} + \frac{1}{b^3} \right) = 12EI \left( \frac{a^3 + b^3}{a^3 b^3} \right)$$

$$\omega_n = \sqrt{\frac{k_{eq}}{m_{eq}}} = \sqrt{\frac{12EI (a^3 + b^3) g}{W a^3 b^3}}$$

[ ASSUMES NO  
ROTATION OF  
FLYWHEEL  
OCCURS ]

## FIXED-FIXED BEAM WITH NON CENTER TRANSVERSE POINT LOAD.

Solution -

Transverse (OPTION 2)

$$\text{Deflection } y = \frac{Pb^2}{6EI\ell^3} \left[ (2b-3\ell)a^3 + 3\ell(\ell-b)a^2 \right]$$

$$y = \frac{Pa^2b^2}{6EI\ell^3} \left[ 2ab - 3a\ell + 3\ell^2 - 3\ell b \right]$$

$$y = \frac{Pa^3b^3}{3EI(a+b)^3}$$

$$k_{eq} = \frac{P}{y} = \frac{3EI(a+b)^3}{a^3b^3} \quad m_{eq} = \frac{W}{g}$$

$$\omega_n^* = \sqrt{\frac{k_{eq}}{m_{eq}}} = \sqrt{\frac{3EI(a+b)^3 g}{a^3 b^3 W}} \quad \text{when } I = \frac{\pi d^4}{64}$$

THIS APPROACH ALLOWS ROTATION TO  $k_{eq}$  CALCULATION BUT NOT IN  $m_{eq}$

Torsional:

$$k_{eq} = (k_1)_t + (k_2)_t$$

$$k_{eq} = \frac{GJ}{a} + \frac{GJ}{b} = \frac{GJ(a+b)}{ab}$$

$$\text{but } J = \frac{\pi d^4}{32}$$

$$k_{eq} = \frac{G\pi d^4}{32} \frac{(a+b)}{ab}$$

$$m_{eq} = I_0 = mk^2 = mr^2$$

$r = \text{radius of gyration}$

$$m_{eq} = \frac{W}{g} r^2$$

$$\omega_n = \sqrt{\frac{k_{eq}}{m_{eq}}} = \sqrt{\frac{G\pi d^4 (a+b)g}{32 ab W r^2}}$$

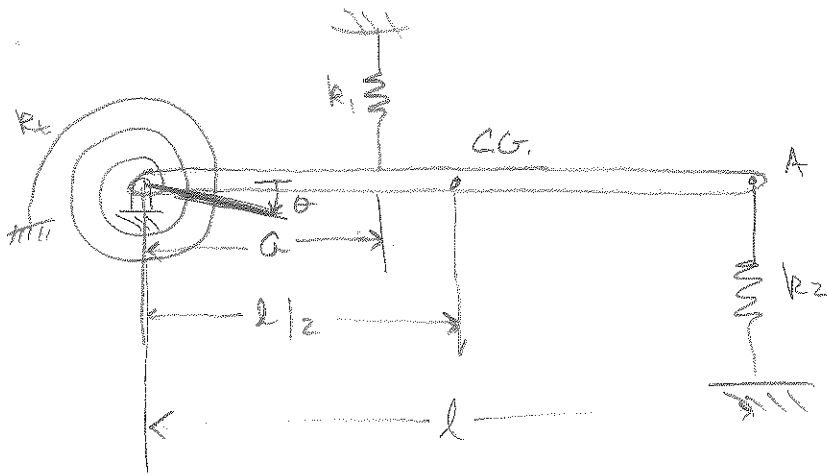
Longitudinal :

$$k_{eq} = k_1 + k_2 = \frac{AE}{a} + \frac{AE}{b} = \frac{AE(a+b)}{ab}$$

$$m_{eq} = W/g$$

$$\omega_n = \sqrt{\frac{k_{eq}}{m_{eq}}} = \sqrt{\frac{AE(a+b)g}{abW}}$$

$$\text{where } A = \frac{\pi d^2}{4}$$



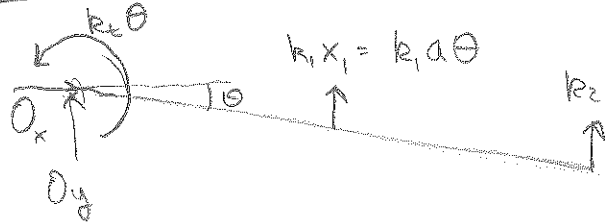
MASS  $OA = m$

USE NEWTON'S 2ND LAW

TO FIND E.O.M.

AND  $\omega_n$

FBD (ASSUME SMALL  $\theta, \dot{\theta}, \ddot{\theta}$ )



$I_0 \ddot{\theta}$



$$I_0 = \bar{I}_G + md^2$$

$$= \frac{1}{12} ml^2 + m \left( \frac{l}{2} \right)^2 = \frac{1}{3} ml^2$$

$$\sum M_0 = \sum M_{\text{eff}}$$

$$k_t \theta + (k_1 a \theta) a + (k_2 l \theta) l = -I_0 \ddot{\theta}$$

E.O.M.

$$\left[ \frac{1}{3} ml^2 \ddot{\theta} + (k_2 + k_1 a^2 + k_2 l^2) \theta = 0 \right]$$

$$\omega_n = \sqrt{\frac{3(k_t + k_1 a^2 + k_2 l^2)}{ml^2}}$$