

Name: KEY

PrEPS Practice Exam 2

Fall 2017

3. The pendulum consists of a disk of weight W_1 and a slender rod of weight W_2 .

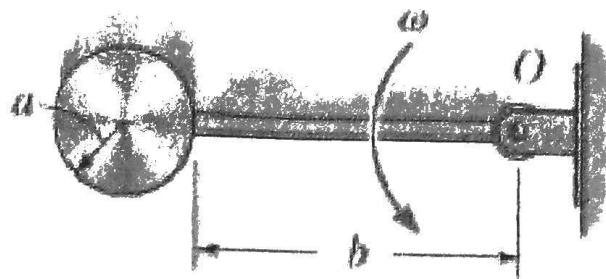
$$W_1 = 15 \text{ lb}$$

$$W_2 = 10 \text{ lb}$$

$$a = 0.75 \text{ ft}$$

$$b = 3 \text{ ft}$$

$$\omega = 8 \frac{\text{rads}}{\text{s}}$$



Determine:

- A) Knowing the disk and the rod are rigidly attached, find the mass moment of Inertia about point O.
- B) Determine the horizontal and vertical components of reaction that the pin O exerts on the rod just as it passes the horizontal position, at which time its angular velocity is ω .

a) $I_O = I_{O, \text{rod}} + I_{O, \text{disk}}$

$$I_{O, \text{rod}} = I_{\text{rod, end}} = \frac{1}{3} m L^2 = \frac{1}{3g} W_2 b^2$$

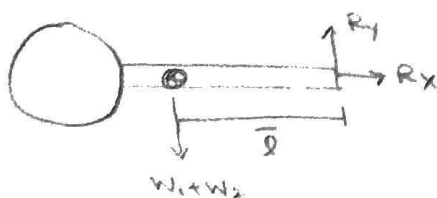
$$I_{O, \text{disk}} = I_{\text{disk}} + m d^2 = \frac{1}{2} m r^2 + m d^2 = \frac{1}{2g} W_1 a^2 + \frac{W_1}{g} (a+b)^2$$

$$I_O = \frac{1}{g} \left[\frac{W_2 b^2}{3} + \frac{W_1 a^2}{2} + W_1 (a+b)^2 \right]$$

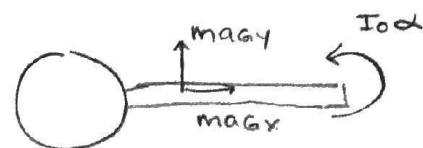
$$I_O = 7.613 \text{ slug} \cdot \text{ft}^2$$

b)

FBD



KBD



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$$(1) \Sigma F_x: R_x = m a_{Gx}$$

$$(2) \Sigma F_y: R_y - W_1 - W_2 = m a_{Gy}$$

$$(3) \Sigma M_O: (W_1 + W_2) \bar{l} = I_O \alpha$$

$$\bar{l} = \frac{\frac{W_1 b}{2} + W_1 (a+b)}{W_1 + W_2}$$

$$\bar{l} = 2.85 \text{ ft}$$

3 equations, 5 unknowns ($a_{Gx}, a_{Gy}, R_y, R_x, \alpha$)
Kinematics for last two equations

$$\vec{a}_G = \vec{a}_O + \vec{\omega} \times \vec{r}_{G/O} - \omega^2 \vec{r}_{G/O}$$

$$a_{Gx} \hat{i} + a_{Gy} \hat{j} = (\alpha \hat{k} \times -\bar{l} \hat{i}) - \omega^2 (-\bar{l} \hat{i})$$

$$a_{Gx} \hat{i} + a_{Gy} \hat{j} = -\alpha \bar{l} \hat{j} + \omega^2 \bar{l} \hat{i}$$

$$(4) \hat{i}: a_{Gx} = \omega^2 \bar{l}$$

$$(5) \hat{j}: a_{Gy} = -\alpha \bar{l}$$

From (1) and (4)

$$R_x = m \omega^2 \bar{l} = \left[\frac{W_1 + W_2}{g} \right] \omega^2 \bar{l}$$

$$R_x = 141.615 \text{ lbs}$$

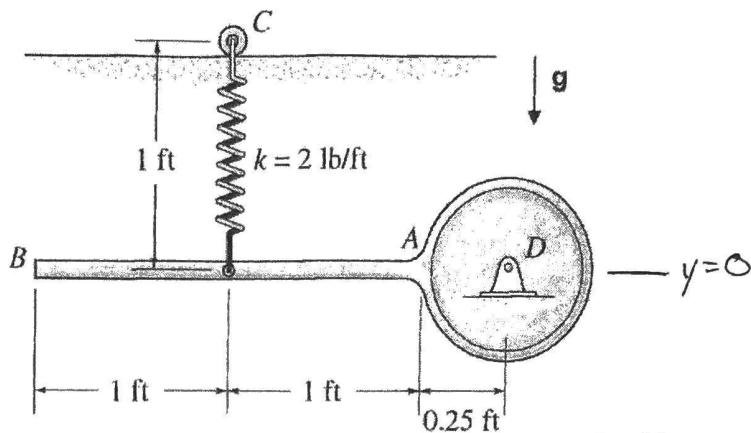
Put (3) into (5) and then (5) into (2)

$$\alpha = \frac{(W_1 + W_2) \bar{l}}{I_O} \quad \alpha = 9.358 \text{ rad/s}^2$$

$$R_y = W_1 + W_2 + \left[\frac{W_1 + W_2}{g} \right] [-\alpha \bar{l}]$$

$$R_y = 4.291 \text{ lbs}$$

2. The pendulum consists of 2-lb rod BA and a 6 lb disk. The spring is stretched 0.3 ft when the rod is horizontal as shown. When the pendulum is released, it rotates about bearing D. The roller at C allows the spring to remain vertical as the rod falls.



- (a) If released from rest, determine the pendulum's angular velocity at the instant the rod becomes vertical.
- (b) Suppose we now add a torsional spring of strength k_t to the bearing at D. The torsional spring provides a restoring moment in much the same way as a linear spring provides a restoring force. It has an associated potential function of $V_t = \frac{1}{2} k_t \theta^2$, where θ is the angular displacement of the rod from its horizontal position. What value should k_t have so that the pendulum comes to rest again after displacing 45° ?

$$W_{rod} = 2 \text{ lb} \quad L_0 = 0.7 \text{ ft} \quad R = 0.25 \text{ ft}$$

$$W_{disk} = 6 \text{ lb} \quad L_{AB} = 2 \text{ ft}$$

$$T_1 + V_1 = T_2 + V_2$$

$$T_1 = 0 \quad V_1 = \frac{1}{2} k \delta_1^2$$

$$T_2 = \frac{1}{2} m v_G^2 + \frac{1}{2} I_G \omega^2 = \frac{1}{2} I_D \omega^2$$

$$I_D = I_{disk,G} + I_{rod,G} + m_{rod} d_{G/B}^2$$

$$I_D = \frac{1}{2} m_{disk} R^2 + \frac{1}{12} m_{rod} L_{AB}^2 + m_{rod} \left(\frac{L_{AB}}{2} + R \right)^2$$

$$V_2 = \frac{1}{2} k \delta_2^2 + m_{rod} g \Delta H_{rod,G}$$

$$\frac{1}{2} k \delta_1^2 = \frac{1}{2} \left[\frac{1}{2} \cdot \frac{W_{disk}}{g} \cdot R^2 + \frac{W_{rod}}{g} \left(\frac{1}{12} L_{AB}^2 + \left(\frac{L_{AB}}{2} + R \right)^2 \right) \right] \omega^2 + \frac{1}{2} k \delta_2^2 + \frac{W_{rod}}{g} g \Delta H_{rod,G}$$

$$\delta_1 = 0.3 \text{ ft}$$

$$\delta_2 = \delta_1 + \frac{L_{AB}}{2} + R = 1.55 \text{ ft}$$

$$\Delta H_{rod} = - \left(\frac{L_{AB}}{2} + R \right) = -1.25 \text{ ft}$$

$$\frac{1}{2} (2 \frac{\text{lb}}{\text{ft}}) ((.3 \text{ ft})^2 - (1.55 \text{ ft})^2) + (2 \text{ lb})(1.25 \text{ ft}) = \frac{1}{2} \left[\frac{1}{2} \cdot \frac{6 \text{ lb}}{32.2 \frac{\text{ft}}{\text{s}^2}} (.25 \text{ ft})^2 + \frac{2 \text{ lb}}{32.2 \frac{\text{ft}}{\text{s}^2}} \left(\frac{(2 \text{ ft})^2}{12} + \left(\frac{2 \text{ ft}}{2} + .25 \text{ ft} \right)^2 \right) \right] \omega^2$$

$$.1875 = .0618 \cdot \omega^2$$

$$\boxed{\omega = 1.742 \frac{\text{rad}}{\text{s}}}$$

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$$T_1 + U_1 = T_2 + U_2$$

$$T_1 = 0 \quad T_2 = 0$$

$$U_1 = \frac{1}{2} k \delta_1^2 \quad U_2 = \frac{1}{2} k \delta_2^2 + \frac{1}{2} k_t \theta^2 + m_{\text{rod}} g \Delta H_{\text{rod},6}$$

$$\delta_1 = 0.3 \text{ ft}$$

$$\delta_2 = 0.3 \text{ ft} + \left(\frac{L_{AB}}{2} + R \right) \sin 45^\circ = 1.184 \text{ ft}$$

$$\Delta H_{\text{rod},6} = - \left(\frac{L_{AB}}{2} + R \right) \sin 45^\circ = -0.8838 \text{ ft}$$

$$\frac{1}{2} k \delta_1^2 = \frac{1}{2} k \delta_2^2 + \frac{1}{2} k_t \theta^2 + W_{\text{rod}} \Delta H_{\text{rod},6}$$

$$\frac{\frac{1}{2} k (\delta_1^2 - \delta_2^2) - W_{\text{rod}} \Delta H_{\text{rod},6}}{\frac{1}{2} \theta^2} = k_t$$

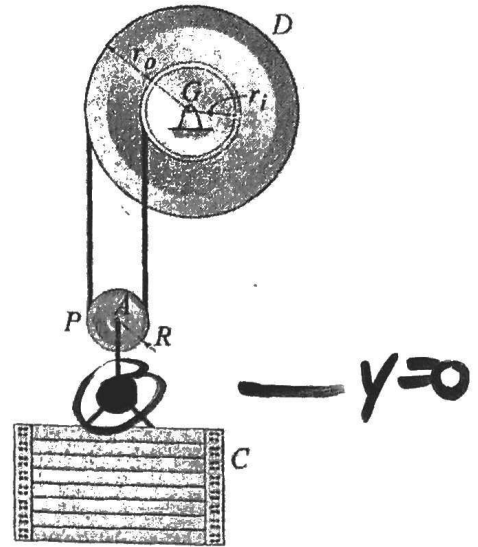
$$k_t = \frac{(2 \frac{16}{\text{ft}}) (10.3 \text{ ft})^2 - (1.184 \text{ ft})^2 - 2(216)(-0.8838 \text{ ft})}{(\pi/4 \text{ rad})^2}$$

$$k_t = 1.479 \text{ lb-ft}$$

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3. The double pulley D has a mass of 15kg, center of mass G coinciding with its geometric center, radius of gyration $k_G = 10\text{cm}$, outer radius $r_o = 15\text{cm}$, and inner radius $r_i = 7.5\text{cm}$. It is connected to the pulley P with radius R via a cord of negligible mass that unwinds from the inner and outer spools of the double pulley D . The crate C , which has a mass of 20kg, is released from rest.

Assuming the pulley P has a mass of 1.5 kg and a radius of gyration $k_G = 3.5\text{cm}$, determine the speed of the crate C and the angular velocity of pulley D after the crate has dropped a distance $h = 2\text{m}$



$$m_D = 15\text{ kg} \quad r_o = 15\text{ cm} \quad m_c = 20\text{ kg} \quad k_{G,P} = 3.5\text{ cm}$$

$$k_{G,D} = 10\text{ cm} \quad r_i = 7.5\text{ cm} \quad m_P = 1.5\text{ kg} \quad h = 2\text{ m}$$

$$T_1 + V_1 = T_2 + N_2$$

$$T_1 = 0 \quad T_2 = \frac{1}{2} I_D \omega_D^2 + \frac{1}{2} m_P v_{G,P}^2 + \frac{1}{2} I_P \omega_P^2 + \frac{1}{2} m_C v_C^2$$

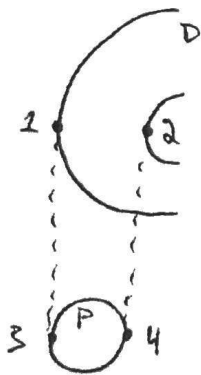
$$V_1 = 0 \quad v_2 = -(m_P + m_C) h g$$

$$I_D = m_D k_{G,D}^2$$

$$I_P = m_P k_{G,P}^2$$

find: $\omega_D \quad \omega_P \quad v_P \quad v_C$

$$v_P = v_C \quad \checkmark$$



$$v_1 = \omega_D r_o = v_3 \quad v_2 = \omega_D r_i = v_4$$

$$v_P = \frac{1}{2} (v_3 + v_4)$$

$$v_P = v_3 + \omega_P \times \vec{r}_{P/3} = v_4 + \omega_P \times \vec{r}_{P/4} \stackrel{\pm}{=} \omega_D r_o + \omega_D \cdot \frac{1}{2} (r_o - r_i)$$

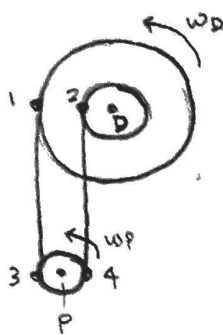
$$= \omega_D (r_i + r_o) \cdot \frac{1}{2}$$

$$v_3 + \omega_P R = v_4 + \omega_P (-R)$$

$$2 \omega_P R = v_4 - v_3$$

$$\omega_P = \frac{v_4 - v_3}{2R} = \frac{\omega_D r_o - \omega_D r_i}{2(\frac{1}{2})(r_o - r_i)} = \frac{\omega_D (r_o - r_i)}{r_o - r_i} = \omega_D$$

Problem 3 kinematics



$$\vec{v}_1 = \vec{v}_D^0 + \vec{\omega}_D \times \vec{r}_{1/D}$$

$$\vec{v}_1 = \omega_D \hat{k} \times -r_o \hat{i}$$

$$\vec{v}_1 = -\omega_D r_o \hat{j}$$

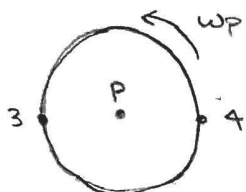
$$\vec{v}_2 = \vec{v}_D^0 + \vec{\omega}_D \times \vec{r}_{2/D}$$

$$\vec{v}_2 = \omega_D \hat{k} \times -r_i \hat{i}$$

$$\vec{v}_2 = -\omega_D r_i \hat{j}$$

$$\vec{v}_1 = \vec{v}_3$$

and $\vec{v}_2 = \vec{v}_4$



Let the distance between 3 and 4 be $r_o - r_i$

$$\vec{v}_4 = \vec{v}_3 + \vec{\omega}_P \times \vec{r}_{4/3}$$

$$-\omega_D r_i \hat{j} = -\omega_D r_o \hat{j} + \omega_P \hat{k} \times (r_o - r_i) \hat{i}$$

$$-\omega_D r_i \hat{j} = -\omega_D r_o \hat{j} + \omega_P (r_o - r_i) \hat{j}$$

$$\omega_D (r_o - r_i) = \omega_P (r_o - r_i)$$

$$\therefore \boxed{\omega_D = \omega_P}$$

$$\vec{v}_P = \vec{v}_3 + \vec{\omega}_P \times \vec{r}_{P/3}$$

$$\vec{v}_P = -\omega_D r_o \hat{j} + \omega_D \hat{k} \times \frac{(r_o - r_i)}{2} \hat{i}$$

$$\vec{v}_P = \left[-\omega_D r_o + \omega_D \frac{(r_o - r_i)}{2} \right] \hat{j}$$

$$\vec{v}_P = \frac{-2\omega_D r_o + \omega_D r_o - \omega_D r_i}{2}$$

$$\boxed{\vec{v}_P = \frac{-\omega_D (r_o + r_i)}{2}}$$

$$\vec{v}_P = \vec{v}_C$$

Knowing $\omega_D = \omega_P$ and $v_P = \frac{\omega_D (r_o + r_i)}{2}$ and $v_C = v_P$

$$0 = T_2 + V_2$$

$$0 = \frac{1}{2} I_O \omega_D^2 + \frac{1}{2} m_P v_P^2 + \frac{1}{2} I_P \omega_P^2 + \frac{1}{2} m_C v_C^2 - (m_P + m_C) h g$$

$$2(m_P + m_C) h g = m_D K_{G,D}^2 \omega_D^2 + m_P \frac{\omega_D^2 (r_o + r_i)^2}{4} + m_P K_{G,P}^2 \omega_D^2 + m_C \frac{\omega_D^2 (r_o + r_i)^2}{4}$$

$$2(m_P + m_C) h g = \omega_D^2 \left[m_D K_{G,D}^2 + m_P K_{G,P}^2 + \frac{(m_P + m_C) (r_o + r_i)^2}{4} \right]$$

$$\omega_D = \sqrt{\frac{2(m_P + m_C) h g}{m_D K_{G,D}^2 + m_P K_{G,P}^2 + \frac{(m_P + m_C) (r_o + r_i)^2}{4}}}$$

$$\omega_D = 44.61 \frac{\text{rad}}{\text{s}}$$

Since $v_C = v_P = \frac{\omega_D (r_o + r_i)}{2}$

$$v_C = 5.019 \text{ m/s}$$